

Pre & post-processor for Laby DRBSF



70R

TURBOMACHMERY ROTORDYNAMICS ANALYSIS

Users Manual

Version 1.1

April 20, 2016

NEGAVIB Christiansburg, VA 24073

Contact information

This program was developed in the mid 1980's for rotor dynamics design evaluation of machinery having toothed labyrinth seals. Options exist in the data entry to estimate the leakage flow entry swirl into the first tooth of the seal. The program compares well to CFD analysis of similar seals but the effects are slightly larger in DYNLAB (LabyDRBSF.exe), hence if the system is stable with DYNLAB coefficients, the machine will likely be stable concerning labyrinth seal effects.

The current version has a new pre & post-processor with new features to assist in the analysis of several options of bladed labyrinth seals. The new frontend (Labyseal.exe) was written by Dr. Wen Jeng Chen and includes a very helpful graphic display of the leak-path and indication of tooth placement.

The latest version is now available as a part of DyRoBeS.

See Appendix E for more information about the program.

Please see Appendix F for the download instructions that will be useful if you decide to purchase the program.

Any problems, bug reports or suggestions regarding this program should be directed to:

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LABYSEAL / LabyDRBSF for labyrinth seal analysis

Introduction to general operation

To execute the program:

- 1. Load the Labyseal setup files as explained in Appendix F.
- 2. Make a run directory for saving outputs and input file
- 3. Click on Labyseal.exe to execute the pre-processor or use the run option in DyRoBeS.
- 4. Input data is saved in filename.LSI
- 5. Bottom of input window shows important results, toggle to other speed cases.
- 6. Program results saved in output file filename.ORG
- 7. Shorter version of current output saved in fixed name file CURRENT.ORG.
- 8. Stiffness and damping values saved in file filename.LSO for use by rotor response programs.

Additional information about the analysis program input detail format can be found in Appendix C of this document

Please note the following:

MuxG input gas viscosity units !!!! are [LBm/(Ft-sec)]

Kinematic viscosity = MuxG / 12/PS*Z*R*TABS [in^2/sec]

Centipoises = MuxG/12/386/1.45E-7 [cp]

_				elect Englis	h or metr	ic units in	itially	
Laby Seal (Calculation - C:	\LabySeal\Exa	mple\MASTE	R_rgkver1.LSI				
Comment: c	ase for co2 com	pressor						
Comment: 2	1 teeth clearand	ce 0.008 series	10 with tooth	on stator == ST 11	1			
Comment: c	heck case cond	itions from origin	al version irgł	< Oct 27 2015 cld	ose			
Labyrinth	Series: Badial	Chambers Befor	e and/or After	the ''Labu''	•	Units: English		onvert Units
-	,		T	the Laby	·	Units: [English		
Tooth Lo	,	on Stator	<u> </u>			12.00	1	
	imber of Teeth:	23		Gas Mol	lecular Weight:	43.86		
Τc	ooth Height (B):	0.0937	in	(Compressibility:	0.7857		
Tooth to To	oth Length (L):	0.0937	in	Ratio of	Specific Heat:	1.6		
Radial	Clearance (C):	0.008	in	Cp at Con:	stant Pressure:	0.37	BTU/LBM	
Sł	haft Radius (R):	1.4375	in		Viscosity:	1.65e-005	LBM/FT/9	SEC
	Rotor Speed:	10832	RPM	Absolute	e Temperature:	770	DEG-R	
First	Critical Speed:	7400	RPM	Upstream	High Pressure:	1800	PSI	
First toot	h for K, C calcul	ation: 9		Low P	ressure at Exit:	938	PSI	New
Last too	th for K, C calcul	ation: 21		Absolute	e Gas Velocity:	252	FT/SEC	Open
Analysis	Type: Non-Syr	nchronous Whirl	and Aero Q C	alculation 🛛 👻	Inlet Swirl:	0.59		· · ·
	1			1	La le l		. 1	Save
To	oth Table	Turbulence Coefficient:		nvergence arameters	Multiple Case/Speed	d Geom		Save As
Summ	arized Results fo	r Selected Case	e, Use << or>>>	to Select				_
	Case Numbe	er: <<- 1	+>>	Effective Ae	eroQ = S	9108 Lbf	Zin	Run
	К = Кхх = Куу	y = -1027	4 Lbf/in	C = Cxx =	= Суу = 🛛 б	6.346 Lbf	-s/in	Develo
	k = Кху = -Кух	(= 14025	5 Lbf/in	с = Сху =	-Cyx = 8	3.949 Lbf	-s/in	Results
	Flow Rate	e = 2.721	Lbm/s	ec SI	CFM = 1	1109		Close

Description of Labyseal features

- Figure 1 Window for Labyseal.exe after execution by clicking Run button Results for speed case 1 showing in lower part of window
- Note: Flow parameter SCFM (Standard cubic feet per min) is calculated for 60 degF and 14.7 psia.

Laby Seal Calculation - C:	\LabySeal\Exar	nple\MASTER_	rgkver1.LSI			×
Comment: case for so2 comp	oressor					
Comment: 21 teeth clearance	e 0.008 series	10 with tooth on	stator == ST 11			
Comment: check case cond	tions from origina	al version irgk C)ct 27 2015 close			
Labyrinth Series: Radial	Chambers Befor	e and/or After th	e ''Laby''	Units: English	▼ Co	nvert Units
Tooth Location: Tooth I	nterlocking 🗸	Starts from:	Stator 👻	,		
Number of Teeth:	23		Gas Molecular Weight:	43.86		
Tooth Height (B):	0.0937	in	Compressibility:	0.7857		
Tooth toTooth Length (L):	0.0937	in	Ratio of Specific Heat:	1.6		
Radial Clearance (C):	0.008	in	Cp at Constant Pressure:	0.37	BTU/LBM/	'DEG-R
Shaft Radius (R):	1.4375	in	Viscosity:	1.65e-005	LBM/FT/S	EC
Rotor Speed:	10832	RPM	Absolute Temperature:	770	DEG-R	
First Critical Speed:	7400	RPM	Upstream High Pressure:		PSI	
First tooth for K, C calcula	ation: 9		Low Pressure at Exit:	938	PSI	New
Last tooth for K, C calcul	ation: 21		Absolute Gas Velocity:		FT/SEC	Open
Analysis Type: Non-Syn	ichronous Whirl	and Aero Q Calc	ulation 🔽 Inlet Swirl:	0.59		C
Tooth Table	Turbulence	Conv	ergence Multiple	Seal		Save
	Coefficients	Para	meters Case/Spee	d Geome	try	Seve As
Summarized Results fo	r Selected Case	,Use << or >> to	Select			
Case Numbe	r: <<- 1	+>>	Effective Aero Q =	Lbf/i	n	Run
К = Кхх = Куу	. =	Lbf/in	C = Cxx = Cyy =	Lbf-s	/in	Results
k = Kxy = -Kyx	:=	Lbf/in	с = Сху = -Сух =	Lbf-s	/in /	riesuits
Flow Rate	=	Lbm/sec	SCFM =			Close

Option of tooth on stator, or rotor, or interlocking. For interlocking only option of first tooth on rotor or stator

Option of synchronous or non-synchronous whirl solution

Figure 2 Program window with interlocking teeth, shown prior to hitting Run

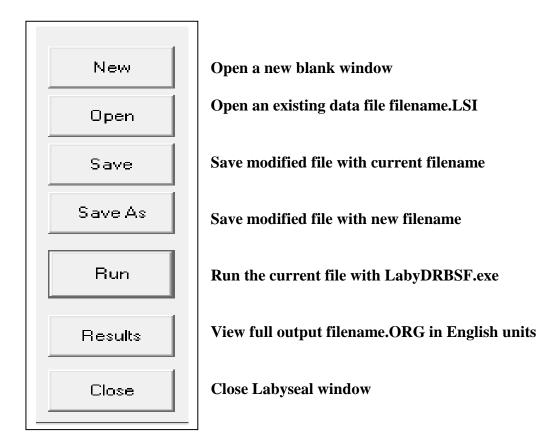
Note: Inlet swirl is inlet swirl ratio, gas swirl velocity / (rotor surface velocity)

ijew Acrobat								
🛃 Laby Seal Calculation - C:\LabySeal\Example\MASTER_rgkver1.LSI								
Comment: case for co2 compressor								
Comment: 21 teeth clearance 0.008 series 10 with tooth on stator == ST 11								
Comment: check case conditions from original version rgk Oct 27 2015 close								
Labyrinth Series: Radial Chambers Before and/or After the "Laby" Units: Metrics Convert Units								
Tooth Location: Tooth on Stator								
Number of Teeth: 23 Gas Molecular Weight: 43.86								
Tooth Height (B): 2.37998 mm Compressibility: 0.7857								
Tooth toTooth Length (L): 2.37998 mm Ratio of Specific Heat: 1.6								
Radial Clearance (C): 0.2032 mm Cp at Constant Pressure: 1.548117154 kJ/kg-K								
Shaft Radius (R): 36.5125 mm Viscosity: 24.55357142 microPa-s								
Rotor Speed: 10832 RPM Absolute Temperature: 427.7777777 DEG-K								
First Critical Speed: 7400 RPM Upstream High Pressure: 12.410566351 MPa								
First tooth for K, C calculation: 9 Low Pressure at Exit: 6.467284023: MPa New								
Last tooth for K, C calculation: 21 Absolute Gas Velocity: 76.8096 M/SEC Open								
Analysis Type: Non-Synchronous Whirl and Aero Q Calculation 🚽 Inlet Swiri: 0.59								
Save								
Tooth Table Turbulence Coefficients Convergence Parameters Multiple Case/Speed Seal Geometry								
Summarized Results for Selected Case, Use << or >> to Select								
Case Number: << · 1 +>> Effective Aero Q = 1595.05 N/mm Run								
K = Kxx = Kyy = -1799.25 N/mm C = Cxx = Cyy = 1.11135 N-s/mm								
k = Kxy = -Kyx = 2456.15 N/mm c = Cxy = -Cyx = 1.56721 N-s/mm								
Flow Rate = 1.23422 kg/sec m^3/min = 31.4034 Close								

Option to elect English or metric units initially

Figure 3 Example of unit conversion of the file shown in Figure 1 using the metric button option noted by arrow above.

Buttons on right side of data entry window are used for file operations



The horizontal buttons in Labyseal window will open supporting additional data entry options

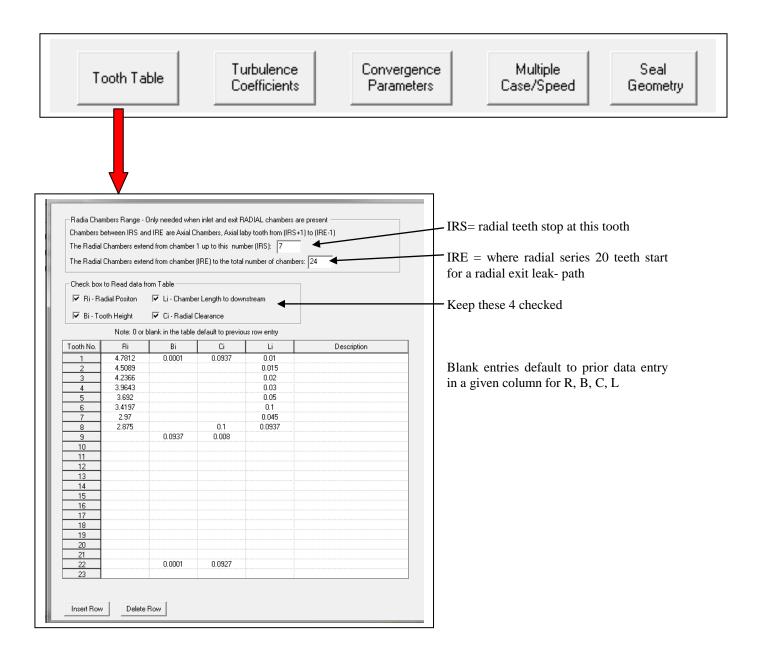


Table input window for teeth radius R, tooth height B, radial clearance C. and chamber length to right of tooth L, plus a description area to describe the teeth if desired.

	vergence Multiple Seal ameters Case/Speed Geometry
Turbulence Coefficients U Default Values Check box to Read data from Table YNR: 0.073 YNS: 0.073 YMR: 0.25 YMS: 0.073 YMR: 0.25 YMS: 0.073 YNR: 0.25 YMS: 0.073 YNR: 0.25 YMS: 0.073 YNR: 0.25 YMS: 0.079 YNR Raito Factor: 1 YMR Cross Flow Option YNS Raito Factor: 1 YMS Cross Flow Factor: 0 YNS Raito Factor: 1 YMS YMS 1 0.079 -0.25 0.079 -0.25 2 0.073 -0.25 0.079 -0.25 3	Normally these stay as given YNR YNS are global scaling factors YMR YMS give crossflow when == 0 Value of 1 will stop crossflow effects Blanks default to prior column value when the program Run button is clicked. So this table has all teeth with same coefficients.

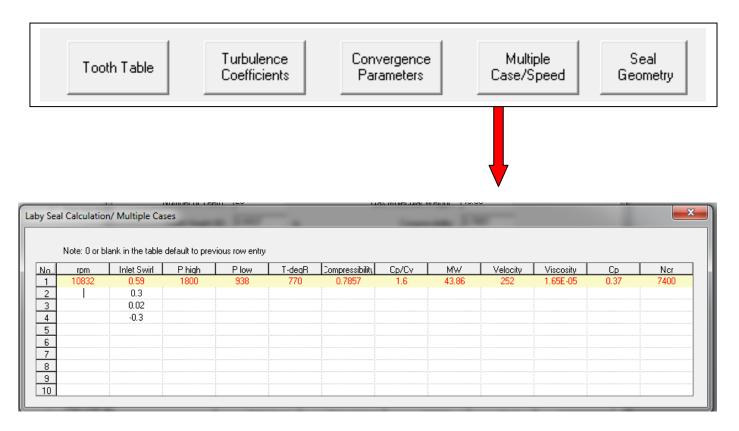
Table input window for turbulence parameters and additional keys in upper section

- Blasius pipe flow factors are N = 0.033 and M = -0.25
- honeycomb flow factors are N = 0.015 and M = -0.025
- default values are per YAMADA, ie., N = 0.079 and M = -0.25

Tooth Table	Turbulence Coefficients Convergence Parameters Case/Speed Geometry
Normal values ar	re given automatically
ſ	Convergence and Control Parameters
T Jł	Print the Maxtrix for Dynamic Calculations OK
2	Maximum number of Solution Iterations: 50
5 U	Velocity Tolerance: 0.0001
51	Pressure Tolerance: 0.01
51	Mass Flow Tolerance: 0.01
	Flow Correction Factor: 1
- - -	Stepped Shafting Step Shaft Pressure Calculation Step Shaft Swirl Iterations: 3

Convergence and control parameters

Speed case option



Max of 10, zero defaults to last speed value entered, value required in at least one column to activate increase of speed case count. This example has 4 speed case conditions.

RPM Swirl ratio Pin Pout TdegR Z cp/cv MW Inlet vel. Viscosity Cp Ncr

Rotor speed rev per minute

Inlet swirl ratio, tangential velocity over surface speed of rotor at tooth 1

Inlet pressure

Discharge pressure

Absolute temperature of gas, average in seal

Compressibility of gas, Z

Ratio of cp / cv for gas in seal

Moleweight of gas in seal

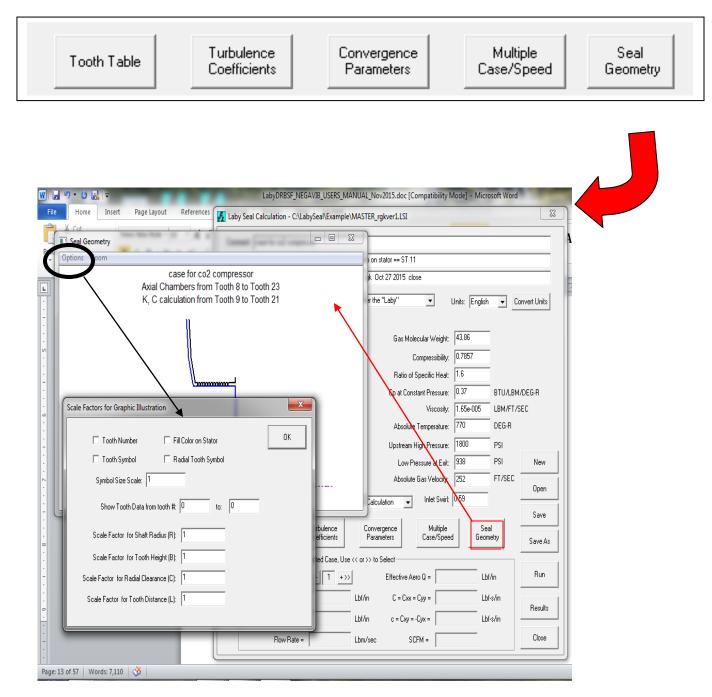
Inlet gas velocity at chamber in front of tooth 1

Viscosity of gas in units specified by unit selection

Specific heat at constant pressure, Cp

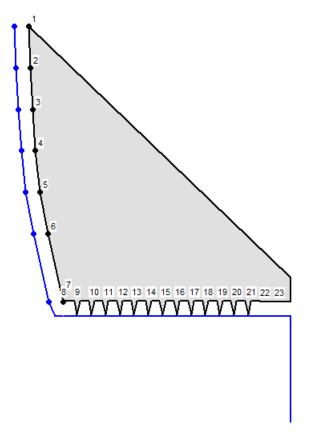
Ncr is the critical speed of shaft for approximate evaluation of Effective Q or Effective C

Option to examine seal leak-path and tooth placement



Window showing when Options selected on Seal Geometry window. To change scale select the Zoom option.

Examples of series and tooth type selection



Series 10 type 1, on stator teeth

Axial Chambers from Tooth 1 to Tooth 25 K, C calculation from Tooth 9 to Tooth 17

1		3		5		7		9		11		13		15	_	17		19	1	21		23	25	5
								¥.	٨.	¥.	٨.	V.	٨.	¥.	Ľ	V.								-
	2		4		6		8		10		12		14		16		18	2	20	1	22	2	4	

Series 00 type 3, interlocking, first on stator (odd tooth)

Axial Chambers from Tooth 1 to Tooth 25 K, C calculation from Tooth 9 to Tooth 17

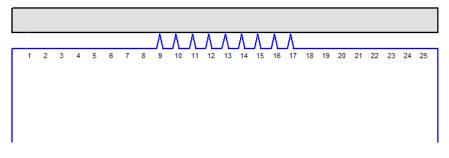
	2	4	6	8		0 12				3 20) 22	24	
					λY	λY	λY	ΛY	<u>ν</u>				
1		3	5	7	9	11	13	15	17	19	21	23	25

Series 00 type 3, interlocking, first on rotor (odd tooth)

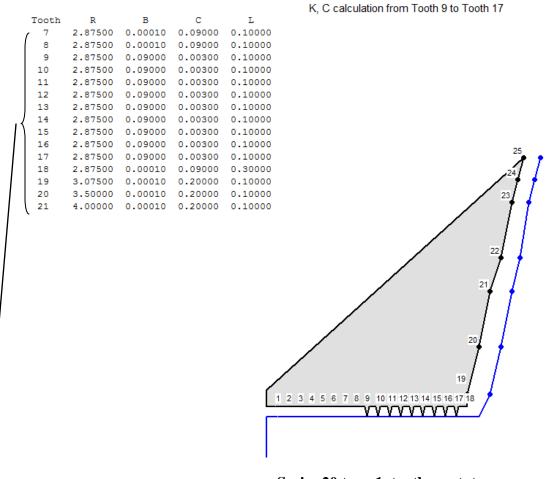
Axial Chambers from Tooth 1 to Tooth 25 K, C calculation from Tooth 9 to Tooth 17

1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25

Series 00 type 1, tooth on stator



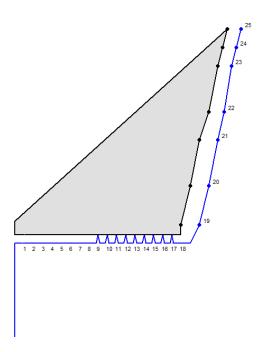
Series 00 type 2, tooth on rotor



Series 20 type 1, tooth on stator

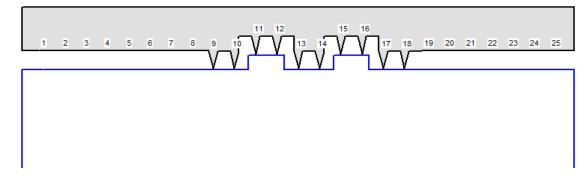
Axial Chambers from Tooth 1 to Tooth 18

▶ Option on Seal Geometry, to list selected teeth values, activated for 7 – 21 in this example.



Series 20 type 2 tooth on rotor

	Example	e from Dr. I	Kirk all axi	ial series 00 type 1 tooth on statorr stepped shaft, covered inlet and outlet Axial Chambers from Tooth 1 to Tooth 25 K. C calculation from Tooth 9 to Tooth 18
Tooth	R	в	с	L
9	2.87500	0.09000	0.00800	0.10000
10	2.87500	0.09000	0.00800	0.10000
11	2.95000	0.09000	0.00800	0.10000
12	2.95000	0.09000	0.00800	0.10000
13	2.87500	0.09000	0.00800	0.10000
14	2.87500	0.09000	0.00800	0.10000
15	2.95000	0.09000	0.00800	0.10000
16	2.95000	0.09000	0.00800	0.10000
17	2.87500	0.09000	0.00800	0.10000
18	2.87500	0.09000	0.00800	0.10000



Series 00 type 1 tooth on stator

Example original version data file for 21 tooth analysis for 13 tooth balance drum seal:

----- CASE FOR CO2 COMPRESSOR THAT WENT UNSTABLE R.G.KIRK DATA FOR SEAL WITH 21 TEETH CLEARANCE AT 0.00800 IBM-PC VERSION ROTOR SPEED-RPM;NSPD;TYPE WHIRL,0=SYN,1=NON;SPD = 10832.000 4 0 RAD OPT;SERIES;TYP 1 STA./2 ROT/3 I-LOCK;STEP;I;IRS=8ST=11 IS=1 IB=1 RADIAL OPT.:NUMBER OF TEETH; PLUS RANGE FOR KXY;IRE= 0NT=21NS= 9NE=21 TOOTH HEIGHT(IF KEY=1,INSERT LINE(S) 7G10.2)KEY = 1B=.01 .01 0.01 0.0937 0.0937 TOOTH SPACING (KEY=1,INSERT LINE(S) 7G10.2)KEY = 1L= 0.01 0.01 0.0 0.0937 0.0937 .0937 TOOTH CLEARANCE(KEY=1,INSERT LINE(S) 7G10.2)KEY = 1C= 0.0937 0.0937 .17500 0.008 .00800 SHAFT RADIUS (KEY=1,INSERT LINE(S) 7G10.2)KEY = 1RS 4.7812 4.7812 4.5089 4.2366 3.9643 3.6920 3.4197 3.1473 2.875 2.875 SURFACE CONSTANT(DEFAULT=0.079);*F5.1;KEY YNR = .079 1.0INR=1 .079 .079 .079 SURFACE CONSTANT(DEFAULT=-0.25);XFLOWI1;KEY YMR = -0.25XFR 0IMR=1 -0.25 -0.25 -0.25 SURFACE CONSTANT(DEFAULT=0.079);*F5.1:KEY YNS = .079 1.0INS=1 .079 .079 .079 SURFACE CONSTANT(DEFAULT=-0.25);XFLOWI1;KEY YMS = -0.25XFS 0IMS=1 -0.25 -0.25 -0.25 GAS SWIRL AT INLET(PERCENT SPD); NCR-RPM; SWIRL = 0.59NC 7400.0 GAS PRESSURE AT INLET; GAS VELOCITY (FT/SEC) PS = 1800.00 V 252.0 GAS PRESSURE AT EXIT ; PE = 938.00 MW = 43.86GAS MOLEWEIGHT GAS TEMPERATURE TABS = 770.00

GAS COMPRESSIBILITY Z = 0.7857GAS RATIO OF SPECIFIC HEATS; AND CP 1.60CP 0.37 GAMMA = GAS VISCOSITY MU * G (LBM/FT/SEC) MUXG = 1.65E-05MASS FLOW (LB/SEC)(IF P(I)>0); FCFAC,F5.1: MDOT = 0.00 0.0 CHAMBER PRESSURES 7G10.2 PER LINE(IF KEY=1) KEY = 0 CHAMBER TEMP. 7G10.2 (IF KEY=1:-2=VARA:0=TABS) = -2 MAX SOLUTION ITERATIONS, VC, MDOT(DEFAULT=50)ITER = 50 TOLERANCE DFAULT = 1E-4*RS*OMEGA TOLERV = 1.0E-2PRESSURE TOLERANCE (DEFAULT = 0.01) 10.0 TOLERP = MASS FLOW TOLERANCE (DEFAULT = 0.0001) TOLERP = .10 PRINTMATRIX SETUP (= 1 TO PRINT ALL) KEYDMP = 0 (IF NSPD>11NPUT NSPD LINES AS FOLLOWS(6G10.2,2F5.1)0 DEFAULTS INIT. V. **SPEED** SWIRL PS PE TABS Z GAMMA MW 10832.000 0.59 1800.00 938.00 770.00 0.7857 1.60 43.86 0.0 0.3 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.02 0.0 0.0 0.0 0.0 0.0 0.0 0.0 -0.30 0.0 0.0 0.0 0.0 0.0 0.0

New data file structure MASTER_rgkver1.LSI for LabyDRBSF:

This file is created by Labyseal.exe frontend.

2.0 // Version Number case for co2 compressor 21 teeth clearance 0.008 series 10 with tooth on stator == ST 11 check case conditions from original version rgk Oct 27 2015 close 10832 00000000 // Speed rpm, // units 0 == English 1 == metric 1 00000000 // Whirl Type 11 1 0 0 0 0 0 0 0 // Tooth Series + Tooth Location, // Interlocking start key 1 stator 0 rotor 23 00000000 // # of Teeth. NT 9 00000000 // 1st Tooth. NS 21 00000000 // Last Tooth, NE 0.0937 000000000 // Tooth Height, B 0.0937 000000000 // Chamber Length, L 0.008 00000000 // Radial Clr, C 1.4375 00000000 // Shaft Radius, R for tooth 1 0.59 00000000 // Swirl Inlet at region ahead of tooth 1 7400 00000000 // 1st Critical Speed 1800 00000000 // High Pressure 252 00000000 // Gas Velocity 938 00000000 // Low Pressure 43.86 00000000 // Mole Weight 770 00000000 // Absolute T 0.7857 000000000 // Compressibility 1.6 00000000 // ratio specific heat 0.37 000000000 // Cp 1.65E-005 0000000000 // Viscosity 1 00000000 // Flow Correction Factor 0.01 000000000 // Tolerance 4 00000000 // # of cases 7 00000000 // IRS 1 00000000 // ISTEP Step Shaft Press Calc 3 00000000 // IISP 24 00000000 // IRE 1 00000000 // KEYB 1 00000000 // KEYL 1 00000000 // KEYC 1 00000000 // KEYR 0.079 000000000 // YNRO 1 00000000 // RRFA 1 00000000 // INR KEY -0.25 000000000 // YMR 0 00000000 // KRRAX 1 000000000 // IMR KEY

0.079 000000000 // YNSO 1 00000000 // SRFA 1 00000000 // INS KEY -0.25 000000000 // YMS 0 00000000 // KRSAX 1 00000000 // IMS KEY 50 00000000 // No. Iteration 0.0001 000000000 // Tolerance Velocity 0.01 00000000 // Tolerance Mass Flow 0 00000000 // PRINT MATRIX, Table R, B, L, C below 1 4.7812 0.0001 0.01 0.0937 0 0 0 0 0 0 // Table input for teeth, NT lines 2 4.5089 0.0001 0.015 0.0937 0 0 0 0 0 0 3 4.2366 0.0001 0.02 0.0937 0 0 0 0 0 0 4 3.9643 0.0001 0.03 0.0937 0 0 0 0 0 0 5 3.692 0.0001 0.05 0.0937 0 0 0 0 0 0 6 3.4197 0.0001 0.1 0.0937 0 0 0 0 0 0 7 2.97 0.0001 0.045 0.0937 0 0 0 0 0 0 8 2.875 0.0001 0.0937 0.1 0 0 0 0 0 0 9 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 10 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 11 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 12 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 13 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 14 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 15 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 16 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 17 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 18 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 19 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 20 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 21 2.875 0.0937 0.0937 0.008 0 0 0 0 0 0 22 2.875 0.0001 0.0937 0.0927 0 0 0 0 0 0 23 2.875 0.0001 0.0937 0.0927 0 0 0 0 0 0 10.079 - 0.250.079 - 0.2500000 // Table for turbulent shear flow parameters, NT lines 2 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 3 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 4 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 5 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 6 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 7 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 8 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 9 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 10 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 11 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 12 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0 13 0.079 -0.25 0.079 -0.25 0 0 0 0 0 0

Appendix - A

List of files provided with LABYSEAL

MASTER_rgkver1.LSI-Example data file for leak path and seal modelMASTER_rgkver1.LSO-Example output file from MASTER_rgkver1.LSI runCURRENT.ORG-Example summary file for MASTER_rgkver1.LSI	-	- - -	
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Appendix – B

LABYRINTH TYPES and NOMENCLATURE for DYNLAB

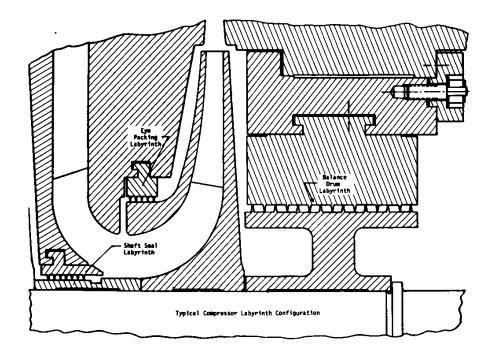


Figure B-1a Typical Compressor Labyrinth Seal Configuration

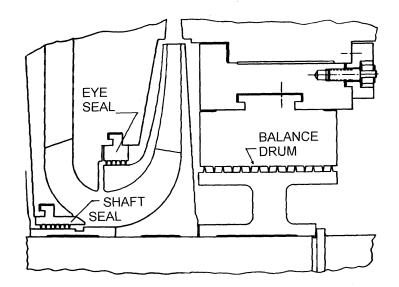


Figure B-1b Typical Compressor Labyrinth Seal Configuration

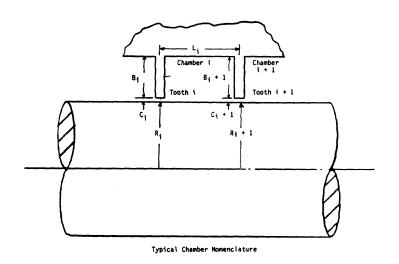


Figure B-2 Typical Chamber Nomenclature

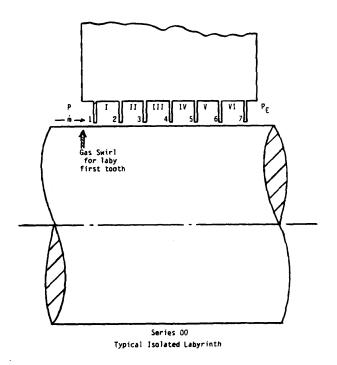
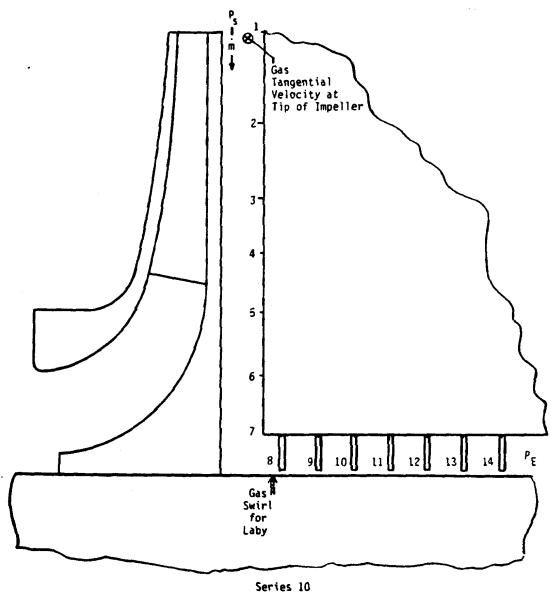
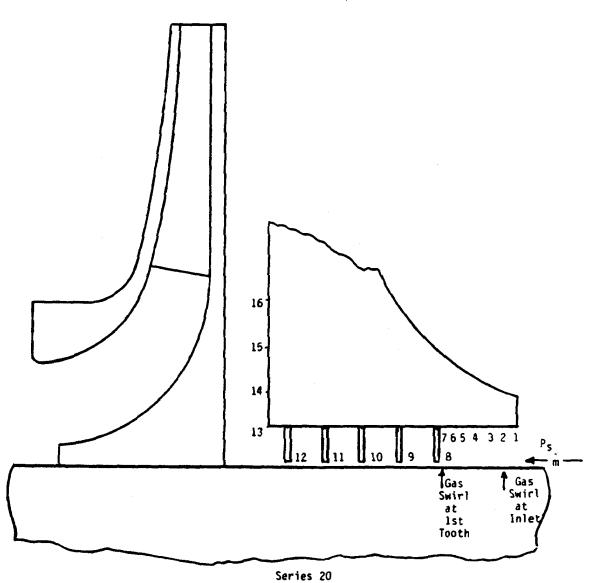


Figure B-3 Typical Series 00 Seal



Typical System for Balance Drum Configuration

Figure B-4 Typical Series 10 Shaft Seal



Typical System for Shaft Seal Labyrinth

Figure B-5 Typical Series 20 Shaft Seal

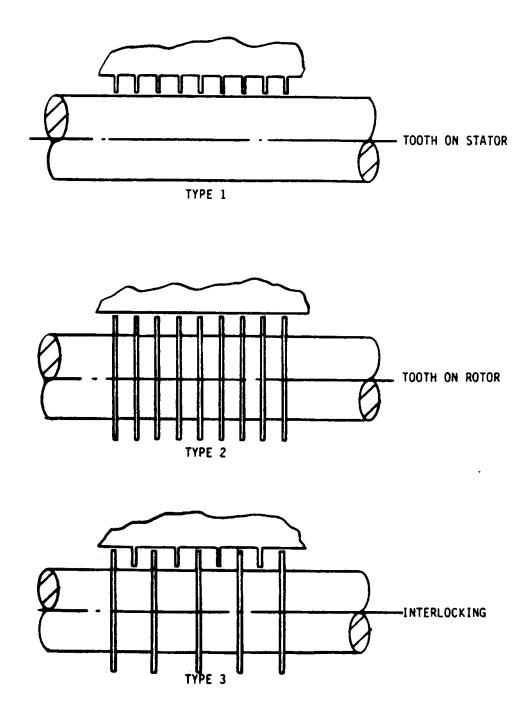
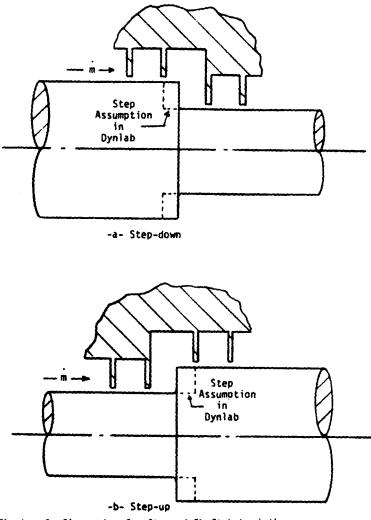


Figure B-6 Type Seal Designation for DYNLAB



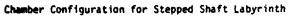


Figure B-7 Typical Step Assumption in DYNLAB

Appendix – C

USERS DETAIL MANUAL FOR THE PROGRAM DYNLAB

A PROGRAM FOR THE ANALYSIS OF

LABYRINTH SEALS

The following example file and comments are given to allow the program DYNLAB to be used

to calculate the dynamic stiffness and damping of labyrinth type gas seals, without the use of the

frontend. This data file can be used to generate information necessary for stability calculations.

Typical Data File (exact column format has been lost in this word file)

```
----- CASE FOR CO2 COMPRESSOR THAT WENT UNSTABLE
                                                      R.G.KIRK
  DATA FOR SEAL WITH 21 TEETH CLEARANCE AT 0.00800 IBM-PC VERSION
  ______
ROTOR SPEED-RPM;NSPD;TYPE WHIRL,0=SYN,1=NON;SPD = 10832.000 4 0
RAD OPT:SERIES:TYP 1 STA./2 ROT/3 I-LOCK:STEP:I:IRS=8ST=11 IS=1 IB=1
RADIAL OPT.;NUMBER OF TEETH; PLUS RANGE FOR KXY;IRE= 0NT=21NS= 9NE=21
TOOTH HEIGHT(IF KEY=1,INSERT LINE(S) 7G10.2)KEY =
                                                   1B=
                                                         .01
   .01
  0.01 0.0937
 0.0937
TOOTH SPACING (KEY=1,INSERT LINE(S) 7G10.2)KEY = 1L=
                                                        0.01
  0.01
                               0.0
 0.0937 0.0937
 .0937
TOOTH CLEARANCE(KEY=1,INSERT LINE(S) 7G10.2)KEY = 1C= 0.0937
 0.0937
 .17500
        0.008
 .00800
SHAFT RADIUS (KEY=1,INSERT LINE(S) 7G10.2)KEY =
                                                  1RS 4.7812
 4.7812 4.5089 4.2366 3.9643 3.6920 3.4197 3.1473
 2.875
 2.875
SURFACE CONSTANT(DEFAULT=0.079);*F5.1;KEY YNR = .079 1.0INR=1
  .079
  .079
  .079
SURFACE CONSTANT(DEFAULT=-0.25);XFLOWI1;KEY YMR = -0.25XFR 0IMR=1
 -0.25
```

-0.25
-0.25
SURFACE CONSTANT(DEFAULT=0.079);*F5.1;KEY YNS = .079 1.0INS=1
.079
.079
.079
SURFACE CONSTANT(DEFAULT=-0.25);XFLOWI1;KEY YMS = -0.25XFS 0IMS=1
-0.25
-0.25
-0.25
GAS SWIRL AT INLET(PERCENT SPD); NCR-RPM; SWIRL = 0.59NC 7400.0
GAS PRESSURE AT INLET; GAS VELOCITY (FT/SEC) PS = 1800.00 V 252.0
GAS PRESSURE AT EXIT; $PE = 938.00$
GAS MOLEWEIGHT $MW = 43.86$
GAS TEMPERATURE $TABS = 770.00$
GAS COMPRESSIBILITY $Z = 0.7857$
GAS RATIO OF SPECIFIC HEATS; AND CP GAMMA = 1.60CP 0.37
GAS VISCOSITY MU $*$ G (LBM/FT/SEC) MUXG = 1.65E-05
MASS FLOW (LB/SEC)(IF P(I)>0); FCFAC, F5.1: MDOT = $0.00 \ 0.0$
CHAMBER PRESSURES 7G10.2 PER LINE(IF KEY=1) KEY = 0
CHAMBER TEMP. 7G10.2 (IF KEY=1;-2=VARA;0=TABS) = -2
MAX SOLUTION ITERATIONS, VC, MDOT (DEFAULT=50) ITER = 50
TOLERANCE DFAULT = $1E-4*RS*OMEGA$ TOLERV = $1.0E-2$
PRESSURE TOLERANCE (DEFAULT = 0.01) TOLERP = 10.0
MASS FLOW TOLERANCE (DEFAULT = 0.0001) TOLERP = .10
PRINTMATRIX SETUP ($= 1$ TO PRINT ALL) KEYDMP $= 0$
(IF NSPD>11NPUT NSPD LINES AS FOLLOWS(6G10.2,2F5.1)0 DEFAULTS INIT. V.
SPEED SWIRL PS PE TABS Z GAMMA MW
10832.000 0.59
0.0 0.3
0.0 0.02 0.0 0.0 0.0 0.0 0.0 0.0
0.0 -0.30

Brief Explanation of Parameters.

It should be noted that values will default to last value read if zero or blank.

Three comment lines for job identification followed by a line that is for instruction

(see example data file above).

80 COL.	COMMENT	Format ((10A8))

80 COL. COMMENT Format (10A8)

80 COL. COMMENT Format (10A8)

<Blank line>

ROTOR SPEED-RPM; NSPD; TYPE WHIRL, 0=SYN, 1=NON; SPD=XXXX.XNSPXX TY=0

Format (50x, G10.2, I2, 4x, I1) SPD, NSPD, KWTYP

- rotor speed, SPD, and number of cases, NSPD to be run
- limit of $\underline{10}$ cases to be run in one input at this time
- type whirl, KWTYP, for seal solution, 0 = synchronous; 1 = non-synch.
- non-synch whirl is at the system natural frequency as given on SWIRL line

RAD OPT; SERIES;TYP 1 STA./2 ROT/3 I-LOCK;STEP;I;IRS=XXST=YX IS=X IB=X

Format (50x, 3x, I2, 3x, I2, 4x, I1, 4x, I1) IRS, TYPE, ISTEP, IISP

- -- series and type labyrinth;
- series 00 have all axial chambers
- series 10 have radial chambers before and/or after the "laby"
- series 20 have radial chambers after the "laby"
- option for series 10 : radial chambers from tooth 1 to IRS
- default value for IRS = NS on next line
- step shaft pressure calc. option ; IS = 1 = yes, 0 = no
- also the number of iterations on swirl for stepped shafting

and for the temperature calculation procedure

• defaults of IB = 2 for IS = 0 and IB = 3 for IS = 1

RADIAL OPT ; NUMBER OF TEETH; PLUS RANGE FOR KXY; IRE=XXNT=XXNS=XXNE=XX

Format (50x, 3x, I2, 3x, I2, 3x, I2, 3x, I2) IRE, NT, NSKQS, NSKQE

- option for series 10 and 20 : radial chambers for tooth IRE to NT
- default value for IRE = NE
- Number of teeth or blades in labyrinth and equivalent system
- also range of teeth for k,c calcs.
- NS = tooth number for start and NE = last tooth for k,c calc
- current limit of <u>50</u> for number of chambers (teeth) input

Format (50x, I10, 2x, G10.2) KEYB, B

- add lines after control line if KEY = 1
- option allows variable chamber parameters
- zero values will default to last non-zero value
- if KEY = 0 all chambers default to value given by B

TOOTH SPACING (KEY=1,INSERT LINE(s)) KEY = $\underline{0}l$ =

.XX

Format (50x, I10, 2x, G10.2) KEYL, L

- add lines after control line if KEY = 1
- option allows variable chamber parameters
- zero values will default to last non-zero value
- if KEY = 0 all chambers default to value given by L

TOOTH CLEARANCE (KEY=1,INSERT LINE(S)) KEY = $\underline{0}C$ =

.XXX

Format (50x, I10, 2x, G10.2) KEYC, C

- add lines after control line if KEY = 1
- option allows variable chamber parameters
- zero values will default to last non-zero value
- if KEY = 0 all chambers default to value given by C

SHAFT RADIUS (KEY=1, INSERT LINE(S) 7G10.2) KEY = 0RS = XX.XX

Format (50x, I10, 2x, G10.2) KEYR, RS

- add lines after control line if KEY = 1
- option allows variable chamber parameters
- zero values will default to last non-zero value
- if KEY = 0 all chambers default to value given by RS

SURFACE CONSTANT(DEFAULT=0.079); F5.1, KEY YNR = _____079 1.0INR=0 Format (50x, G10.2, F5.1, 4x, I1)

SURFACE CONSTANT(DEFAULT=-0.25) ;XFLOWI1;KEYYMR = ______O.25XFR=0IMR=0 Format (50x, G10.2, 3x, I2, 4x, I1)

SURFACE CONSTANT(DEFAULT=0.079);*F5.1;KEY YNS = <u>.079</u> <u>1.0</u>INS=0 Format (50x, G10.2, F5.1, 4x, I1)

SURFACE CONSTANT (DEFAULT=-0.25);XFLOWI1;KEY YMS = <u>-0.25</u>XFS=<u>0</u>IMS=0 Format (50x, G10.2, 3x, I2, 4x, I1)

- use the given default values on these 4 lines usually
- YNR and YNS have ratio factor as option in cols. 61-65
- YMR and YMS have cross flow factor as option in col. 65; 0 = yes
- YMR and YMS have option to input factors per chamber with KEY=1
- if KEY = 0 all chambers default to value given by KEY line
- zero values will default to last non-zero value
- Blasius pipe flow factors are N = 0.033 and M = -0.25
- honeycomb flow factors are N = 0.015 and M = -0.025
- default values are per YAMADA, ie., N == = 0.079 and M = -0.25

GAS SWIRL AT INLET(PERCENT SPD); NCR-RPM; SWIRL = <u>.XXNC_XXXX.X</u> Format (50x, G10.2, 2x, G10.2) Swirl, NCR

- gas swirl at inlet to first chamber of system, ratio of gas
- angular velocity to that of the rotor.
- system natural frequency and perturbation whirl rate if option
- on speed line = 1: equivalent K-xy calculated with N-cr.
- default value for N-cr and whirl is rotor speed (ie., SPD on line 1)

GAS PRESSURE AT INLET; gas velocity (FT/SEC) PS = XXX.X V XXX.X

Format (50x, G10.2, 2x, G10.2) PS, VABS

- gas static pressure before the first chamber of the system
- gas absolute velocity for calculation of total temperature

GAS PRESSURE AT EXIT	$PE = \underline{XX.X}$
----------------------	-------------------------

Format (50x, G10.2) PE

-- gas static pressure after the last chamber of the system

GAS MOLEWEIGHT	MW =	XX.X
----------------	------	------

Format (50x, G10.2) MW

-- input molecular weight for the gas

GAS TEMPERATURE

Format (50x, G10.2) TABS

- input absolute temperature at inlet
- this can be static temperature if velocity specified on PS line

TABS = XXX.X

GAS COMPRESSIBILITY

Z = <u>.XX</u>

Format (50x, G10.2) Z

• input Z compressibility factor for the gas

 $GAMMA = \underline{X.XXCP} \underline{X.X}$

Format (50x, G10.2, 2x, G10.2) GAMMA, CP

GAS RATIO OF SPECIFIC HEATS; AND CP

- input ratio of specific heats for the gas
- also specific heat at constant pressure, CP

GAS VISCOSITY MU * G (LBM/FT/SEC MUXG = X.XXE-XX

Format (50x, G10.2) MUxG

• <u>input gas viscosity</u> <u>note units !!!!</u>) LBm/(Ft-sec)

Kinematic viscosity = MuxG / 12/PS*Z*R*TABS in^2/sec

Centipoises = MuxG/12/386/1.45E-7 cp

MASS FLOW (LB/SEC) (IF P(I)>0); FCFAC,F5.1: MDOT = X.XX X.X

Format (50x, G10.2, F5.1)

- leave mass flow blank if using pressure solution of program
- flow correction factor can be used to match flow to actual
- flow if this is known for the specific tooth type!!!!

CHAMBER PRESSURES 7G10.2 PER LINE (IF KEY=1) KEY =
$$0$$

• option to input chamber static pressure solution

CHAMBER TEMP. 7G10.2 (IF KEY=1;-2 VTR.;0=TABS) <u>-2</u>

• option to input chamber static temperature solution

MAX SOLUTION INTERATIONS, VC, MDOT (DEFAULT=50) ITER = 50

Format (50x, I10) ITER

• option to tolerances for solutions on next 4 lines						
TOLERANCE DEFAULT = 1E-4*RS*OMEGA	TOLERV =	1.0E-4				
Format (50x, G10.2)						
PRESSURE TOLERANCE (DEFAULT) = $.01$)	TOLERP =	.01				
Format (50x, G10.2)						
MASS FLOW TOLEANCE (DEFAULT = .0001)	TOLERM =	.01				
Format (50x, G10.2)						
PRINT MATRIX SETUP (= 1 TO PRINT ALL)	KEYDMP =	<u>0</u>				
Format(50x, 110)						

Format (50x, I10)

• option to print the matrix for dynamic calcs

<black>

<blank>

IF NSPD>1, INPUT NSPD LINES AS FOLLOWS <12G10.2> ; a 0.0 or blank will default to the value in the data input section of the last value entered for that variable.

- option to run multiple cases of speeds or other parameters
- option to have up to <u>10</u> cases is current limit.

SPEED	SWIRL	PS	PE	TABS	Ζ	GAMMA	MW	VAB	S MUX	G CP I	FNCR
<u>XXXX.X</u>	<u>.XX</u>	XXX.X_	_XXX.X	XXX.X	X.X	X.X	_X.X	XXX	XXXX	XXX	XXXXX

Format (12G10.2)

SPD,SWIRL,PS,PE,TABS,Z,GAMMA,MW,VABS,MUXG,CP,FNCR

APPENDIX D

Example output for CURRENT.ORG Summary file for MASTER_RGK.LSI run with four speed cases considered.

Note:

The four cases have a tip swirl of 0.59, 0.3, 0.02, and -0.3. The swirl predicted at entrance to the balance drum seal is given in the summary table of each run (0.75, 0.59, 0.42, 0.23). This clearly shows how the inlet swirl can be changed from the tip of the wheel down the leak path.

SOLUTION FOR LABYRINTH DYNAMIC STIFFNESS 1 THE FOLLOWING SUMMARY CONDITIONS WERE GIVEN: ROTOR SPEED = 10832.00 RPM Ncr = 7400.00 CPMTOTAL TEMP. = 773.43 DEG-R STATIC TEMP. = 771.15 DEG-R INLET PRES. = 1724.46 PSIA DISCH PRES. = 938.22 PSIA SWIRL FACTOR = .75 DIM. VABS = 252.00 FT/sec NO. TEETH = 13 MUXG = .165E-04 lbm/FT/sec MOLEWEIGHT = 43.86 CP = .37 BTU/LBM/dgR ***<<<< STIFFNESS VALUES FOR TEETH 9 TO 21 >>>**** LABYRINTH DYNAMIC STIFFNESS MATRIX DIRECT STIFFNESS = -10274. LB/IN CROSS STIFFNESS = 14026. LB/IN DIRECT DAMPING = 6.3462 LB-SEC/IN CROSS DAMPING = 8.9493 LB-SEC/IN LABYRINTH DAMPING FACTOR = C-KXY/OMEGW = -11.7536 LB-SEC/IN NEGATIVE WILL DRIVE SYSTEM LABYRINTH EFFECTIVE Q-AERO = KXY-C*OMEGW = 9108.18 LB/IN POSITIVE WILL DRIVE SYSTEM

1 SOLUTION FOR LABYRINTH DYNAMIC STIFFNESS THE FOLLOWING SUMMARY CONDITIONS WERE GIVEN: ROTOR SPEED = 10832.00 RPM Ncr = 7400.00 CPM TOTAL TEMP. = 773.43 DEG-R STATIC TEMP. = 772.03 DEG-R INLET PRES. = 1759.92 PSIA DISCH PRES. = 938.23 PSIA SWIRL FACTOR = .59 DIM. VABS = 252.00 FT/sec NO. TEETH = 13 MUXG = .165E-04 lbm/FT/sec MOLEWEIGHT = 43.86 CP = .37 BTU/LBM/dgR

<<<< STIFFNESS VALUES FOR TEETH 9 TO 21 >>>>*

LABYRINTH DYNAMIC STIFFNESS MATRIX DIRECT STIFFNESS = -9211.1 LB/IN CROSS STIFFNESS = 9459.9 LB/IN DIRECT DAMPING = 5.2822 LB-SEC/IN CROSS DAMPING = 7.6518 LB-SEC/IN LABYRINTH DAMPING FACTOR = C-KXY/OMEGW = -6.92533 LB-SEC/IN NEGATIVE WILL DRIVE SYSTEM LABYRINTH EFFECTIVE Q-AERO = KXY-C*OMEGW = 5366.62 LB/IN POSITIVE WILL DRIVE SYSTEM 1 SOLUTION FOR LABYRINTH DYNAMIC STIFFNESS THE FOLLOWING SUMMARY CONDITIONS WERE GIVEN: ROTOR SPEED = 10832.00 RPM Ncr = 7400.00 CPM TOTAL TEMP. = 773.43 DEG-R STATIC TEMP. = 772.70 DEG-R INLET PRES. = 1782.14 PSIA DISCH PRES. = 938.25 PSIA SWIRL FACTOR = .42 DIM. VABS = 252.00 FT/sec NO. TEETH = 13 MUXG = .165E-04 lbm/FT/sec MOLEWEIGHT = 43.86 CP = .37 BTU/LBM/dgR

<<<< STIFFNESS VALUES FOR TEETH 9 TO 21 >>>>*

LABYRINTH DYNAMIC STIFFNESS MATRIX DIRECT STIFFNESS = -8148.9 LB/IN CROSS STIFFNESS = 5108.9 LB/IN DIRECT DAMPING = 4.4133 LB-SEC/IN CROSS DAMPING = 6.1443 LB-SEC/IN LABYRINTH DAMPING FACTOR = C-KXY/OMEGW = -2.17945 LB-SEC/IN NEGATIVE WILL DRIVE SYSTEM LABYRINTH EFFECTIVE Q-AERO = KXY-C*OMEGW = 1688.91 LB/IN POSITIVE WILL DRIVE SYSTEM

1 SOLUTION FOR LABYRINTH DYNAMIC STIFFNESS THE FOLLOWING SUMMARY CONDITIONS WERE GIVEN: ROTOR SPEED = 10832.00 RPM Ncr = 7400.00 CPM TOTAL TEMP. = 773.43 DEG-R STATIC TEMP. = 773.21 DEG-R INLET PRES. = 1791.45 PSIA DISCH PRES. = 938.25 PSIA SWIRL FACTOR = .23 DIM. VABS = 252.00 FT/sec NO. TEETH = 13 MUXG = .165E-04 lbm/FT/sec MOLEWEIGHT = 43.86 CP = .37 BTU/LBM/dgR

<<<< STIFFNESS VALUES FOR TEETH 9 TO 21 >>>*

LABYRINTH DYNAMIC STIFFNESS MATRIX DIRECT STIFFNESS = -7046.6 LB/IN CROSS STIFFNESS = 419.36 LB/IN DIRECT DAMPING = 3.6737 LB-SEC/IN CROSS DAMPING = 4.1480 LB-SEC/IN LABYRINTH DAMPING FACTOR = C-KXY/OMEGW = 3.13257 LB-SEC/IN NEGATIVE WILL DRIVE SYSTEM LABYRINTH EFFECTIVE Q-AERO = KXY-C*OMEGW = -2427.51 LB/IN POSITIVE WILL DRIVE SYSTEM

APPENDIX E

NEGAVIB Labyrinth Seal Program

US distributor

Rodyn Vibration Analysis, Inc. 1501 Gordon Avenue Charlottesville, VA 22903 <u>434.326.6797</u> <u>Rodyn.com</u>

China

XecaTurbo Technologies Mb: 86-189 1139 6280 Skype: xuefeng0909 <u>Tel:010-8492 4950</u> Fax:010-8492 5792 Offices: Beijing, Shanghai, Xi'an, Changsha, Chengdu Web: <u>www.XecaTurbo.cn</u>

DYNLAB Seal Analysis

References for DYNLAB Theory, Analysis and Results

Rotor Dynamics Analysis Software for PC compatibles:

Gas Labyrinth Seal Analysis Program: DYNLAB (LabyDRBSF)

Cost:: \$ 6,000.00

This program was developed as a design tool for analysis of gas labyrinth seals typical of centrifugal compressor eye, shaft, and balance drum configurations. The major advantage of the labyrinth code is the ability to estimate the seal entrance swirl, given the impeller tip swirl which is a standard output of most any aerodynamic design code. The users manual explains the required input for the Dos program. The program has been tested and selected as the program of choice for toothed labyrinth designs by one major OEM compressor company and one major oil company in the US. The program has been used for 26 years at these companies. The program has been used as a consulting tool for the past 28 years.

The program can evaluate tooth on rotor, on stator or interlocking teeth. The seal can be straight through or stepped. The program input can be adjusted to estimate the influence of honeycomb seal designs with typical cell size. It can estimate actual seal entrance swirl.

A one-day short course is recommended to acquaint the users with the program capabilities and proper use for evaluation of entry swirl. The USA based in-plant course will be given for \$3200 plus travel and living. International plant locations arrangements can be discussed for time and cost.

The compiled program executable and user's manual are included in the purchase price.

Technical Papers on Gas Labyrinth Seal Analysis and Design by R. G. Kirk

Conference Proceedings:

Kirk, R. G., and M. Simpson, "Full Load Testing of an 18,000 H.P. Gas Turbine Driven Centrifugal Compressor for Offshore Platform Service," Bently Nevada Research Corp. Symposium on Stability, NASA CP-2409, 1985.

Kirk, R. G., "Evaluation of Aerodynamic Instability Mechanisms for Centrifugal Compressors," ASME Paper 85-DET-147, presented at 1985 ASME Vibrations Conference, Cincinnati, Ohio, Sept. 11-13, 1985.

Kirk, R. G., "Influence of Disk Leakage Path on Labyrinth Seal Inlet Swirl Ratio," presented at Fourth Workshop on Rotor Dynamic Instability Problems in High Performance Turbomachinery, Texas A&M University, May, 1986, Proceedings, published as NASA- CP2443.

Kirk, R. G., "Labyrinth Seal Analysis for Centrifugal Compressor Design - Theory and Practice," Proceedings of the International Conference on Rotordynamics, Tokyo, September 14-17, 1986.

Kirk, R. G., "A Method for Calculating Labyrinth Seal Inlet Swirl Velocity," Rotating Machinery Dynamics, Vol. 2, ASME DE-Vol. 2, 1987.

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Kirk, R. G., "Evaluation of Liquid and Gas Seals for Improved Design of Turbomachinery," Vibrations in Rotating Machinery, I. Mech. E. Proceedings of 4th International Conference, Edinburgh, Sept. 1988, pp. 387-394.

<u>Guo, Zenglin</u>; Hirano, Toshio; Kirk, R. G., "Application of CFD Analysis for Rotating Machinery, Part 1: Hydrodynamic, Hydrostatic Bearings and Squeeze Film Damper", *Proceedings of ASME Turbo Expo 2003, Power for Land, Sea, and Air,* GT2003-38931, June 16–19, 2003, Atlanta, Georgia, USA.

Hirano, Toshio; Guo, Zenglin; <u>Kirk, R. G.</u>, "Application of CFD Analysis for Rotating Machinery, Part 2: Labyrinth Seal Analysis", In: *Proceedings of ASME Turbo Expo* 2003, Power for Land, Sea, and Air, GT2003-38984, June 16–19, 2003, Atlanta, Georgia, USA.

<u>Kirk, R. G.</u>, Guo, Z., 2004, "Calibration of Labyrinth Seal Bulk Flow Design Analysis Prediction to CFD Simulation Results", IMechE, Eighth International Conference on *Vibrations in Rotating Machinery*, Swansea, UK, Sept. 7-9, 2004, pp 3-12. <u>Guo, Z.</u> and R. G. Kirk, "CFD Evaluation of Turbomachinery Secondary Flow Leak Path and Labyrinth Seal Entry Swirl," ISCORMA3 International Symposium on Stability Control of Rotating Machinery, Cleveland State University, Sept 19-23, 2005.

<u>Kirk, R. G.</u> and Z. Guo, "Labyrinth Seal Forces for a High Speed Centrifugal Compressor Impeller Eye Seal," 2006 IFToMM International Conference, Vienna, Austria, Sept, 2006.

Guo, Z. and <u>R. G. Kirk</u>, "CFD Results of Inlet and Reverse Pumping Flow for Common Labyrinth Seal Leak Path Geometries," ISCORMA-4 International Symposium on Stability Control of Rotating Machinery, Calgary, Alberta, Canada, Aug 27-31, 2007.

<u>Kirk, R.G.</u> and Z. Guo, "Leak Path Geometry Influence on Labyrinth Seal Inlet Swirl Velocity Ratio," BHR 19th International Conference on Fluid Sealing, Poitiers, France, Sept 25-26, 2007.

<u>Kirk, R.G.</u> and Z. Guo, "Influence of Leak Path Friction on Labyrinth Seal Inlet Swirl," STLE Annual Meeting, Cleveland, OH, May 19-21, 2008.

Kirk, R. G. and R. Gao, "Analysis of Rotordynamic Forces for High Inlet Pre-swirl Rate Labyrinth Seals," IMechE, 10th International Conference on *Vibrations in Rotating Machinery*, London, UK, Sept. 11-13, 2012.

Technical Journal Papers

Kirk, R. G., "Evaluation of Aerodynamic Instability Mechanisms for Centrifugal Compressors--Part II: Advanced Analysis," *ASME Journal of Vibration, Acoustics, Stress, and Reliability in Design*, Vol. 110, No. 2, April 1988, pp. 207-212.

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Kirk, R. G., "A Method for Calculating Labyrinth Seal Inlet Swirl Velocity," *Trans. ASME, J. Vibration and Acoustics*, 1990, pp. 380-383.

<u>Guo, Zenglin</u>; Hirano, Toshio; Kirk, R. G., "Application of CFD Analysis for Rotating Machinery, Part 1: Hydrodynamic, Hydrostatic Bearings and Squeeze Film Damper", *ASME Journal of Engineering for Gas Turbines and Power*, 127(2) April, 2005, pp 445-451.

Hirano, Toshio; Guo, Zenglin; <u>Kirk, R. G.</u>, "Application of CFD Analysis for Rotating Machinery, Part 2: Labyrinth Seal Analysis", *ASME Journal of Engineering for Gas Turbines and Power*, 127(4) October, 2005, pp 820-826.

<u>Kirk, R.G.</u> and Z. Guo, "Influence of Leak Path Friction on Labyrinth Seal Inlet Swirl," STLE *Tribology Transactions*, 52(2), 2009, pp139-145. Volume 52, Issue 2 pp. 139-145 | DOI: 10.1080/10402000802105430

Kirk, Gordon and Rui Gao, "Influence of Pre-swirl on Rotordynamic Characteristics of Labyrinth Seals," STLE *Tribology Transactions*, 55(3), pp 357-364, 2012. http://www.tandfonline.com/doi/abs/10.1080/10402004.2012.656880 Volume 55, Issue 3 pp. 357-364 | DOI: 10.1080/10402004.2012.656880

Gao, Rui and Gordon Kirk, "CFD Study on Stepped and Drum Balance Labyrinth Seal," for STLE *Tribology Transactions*, 2013. Best paper in seals 2014-2015, STLE Frank P. Bussick Award

Use of DYNLAB Program for consulting:

Dresser-Rand	Exxon	Mobil	GHH	BP	Mechanical Solutions
AMOCO	ExxonMobil	BP America	RMT	Elliott Company	York
AC Compressor	Solar	DeLaval	Joy	Conmec	Allied Signal
NREC	VECO/ARCO	BOC	Atlas Copco	ODS	0
			-		

Purchase of Source Code: ExxonMobil Dresser-Rand

Companies with .exe version

Dresser-Rand	ExxonMobil	BP America	Lufkin- RMT	Elliott Company	Air Products
GE Oil&Gas	DeMagDeLaval	Rolls-Royce Ener	gy Systems	ODS Denmark	

APPENDIX F

Download Instruction for latest LabySeal program:

- 1. Go to download page http://dyrobes.com/user_download/ password: dyrobes
- 2. Download the latest version: LabySeal_Setup.exe
- 3. Install the program, The installation password: LabySeal_V100

It is recommended to run the setup program – Run As Administrator

- 4 Install the driver, run Install_hasp_driver.BAT from the LabySeal folder Note: If you have installed Dyrobes, you may skip this step.
- 5. If you use hardware protection (HL USB dongle), simply plug the USB dongle and you are ready to run the program.

If you use software protection (SL), run rus_cootg.exe and Collect Information. Save the information file (C2V file) and e-mail us. We will e-mail the license (V2C file) back to you.

For network license, SL is recommended due to its flexibility.

For individual license, either protection, HL or SL, is OK.

- There will be an icon created on your desktop (LabySeal_V1.00), you may start the program from this main menu. Or you may start the program from the Windows – Start.
- 7 Obtain the license file for LabySeal from the distributor
- 8. Update your LabySeal license file. Run rus_cootg.exe with your dongly plugged and select Apply License Update in the second tab with the license file. Then the program will execute.